

A10CO

Series 52 axial piston pump

Size 45,63

Nominal pressure 250 bar Peak pressure 315 bar

Gelan Hydraulics



The A10VO compact unit was designed specifically for mobile applications where it is necessary to boost the A10-inlet with a full flow boost pump, i.e. hydraulic systems for agricultural tractors, running on oil from the transmission (common oil systems). This enables also an effective filtration in the A10-inlet line.

A boost pump with a some what larger displacement can provide extra oil for lubricating purposes.

- Central hydraulic unit for mounting to transmission p.t.o. in mobile applications
- Integrated boost pump
- A10VO main pump with controller
- Filter mounting capability
- Integrated pressure relief valves
- Reduced pipework
- Compact design
- Transmission lubrication port
- Measurement or monitoring ports
- Integrated case drain port inside pilot diameter

Type key

A10C	O	45		/	52		-	V		C		H00	2	
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Hydraulic fluid

Mineral oil (without prefix)

Axial-piston pump

Compact unit in swashplate design, adjustable
Nominal pressure 250 bar, peak pressure 315 bar

A10C

Operating mode

Pump, open-circuit

O

Size

 \triangle Displacement volume $V_{g\max}$ (cm³)

45,63

Control options

Pressure controller	DR			DR
Pressure controller, remote controlled	DRG			DRG
Pressure and flow controller	DFR			DFR
	DFR	1		DFR1
without connection from X to tank				

see RE 92703

Series

52

Direction of rotation

Viewing drive shaft

cw

R

anti-cw

L

Seals

FPM (fluororubber to DIN ISO 1629)

V

Shaft end

Spline shaft SAE, shortened	○	1"	S
Spline shaft SAE, shortened (higher through-drive torque)	○	1"	R
Spline shaft SAE, shortened	●	7/8"	U

Mounting flange

SAE 2-hole

C

Port for service lines

Pressure port B Suction port S	metric thread side ports	07
Pressure port B Suction port S	SAE on opposite sides, metric fixing thread	12

Boost pump

with integrated boost pump

H00

Valves

with integrated pressure relief valves

2

Filtration

Filter installed in boost circuit	●	F
without filter, mounting option only	●	D

● = available

○ = in preparation

Hydraulic fluid

Please refer to our catalogue sheets RE 90220 (mineral oil) and RE 90221 (environmentally friendly hydraulic fluids) for detailed information on selecting hydraulic fluids and on service conditions before the project planning stage.

Operation with environmentally friendly hydraulic fluids may result in modifications to the technical specifications; please consult us if necessary.

Service viscosity range

We recommend selecting the service viscosity (at operating temperature) in the range of

$$v_{\text{opt}} = \text{opt. opt. service viscosity } 16 \dots 36 \text{ mm}^2/\text{s}$$

for optimum efficiency and useful life, in relation to tank temperature (open circuit).

Limiting viscosity range

Service limits are set at the following values:

$$v_{\text{min}} = 10 \text{ mm}^2/\text{s}$$

briefly at max. permissible leakage oil temperature of 90 °C.

$$v_{\text{max}} = 1000 \text{ mm}^2/\text{s}$$

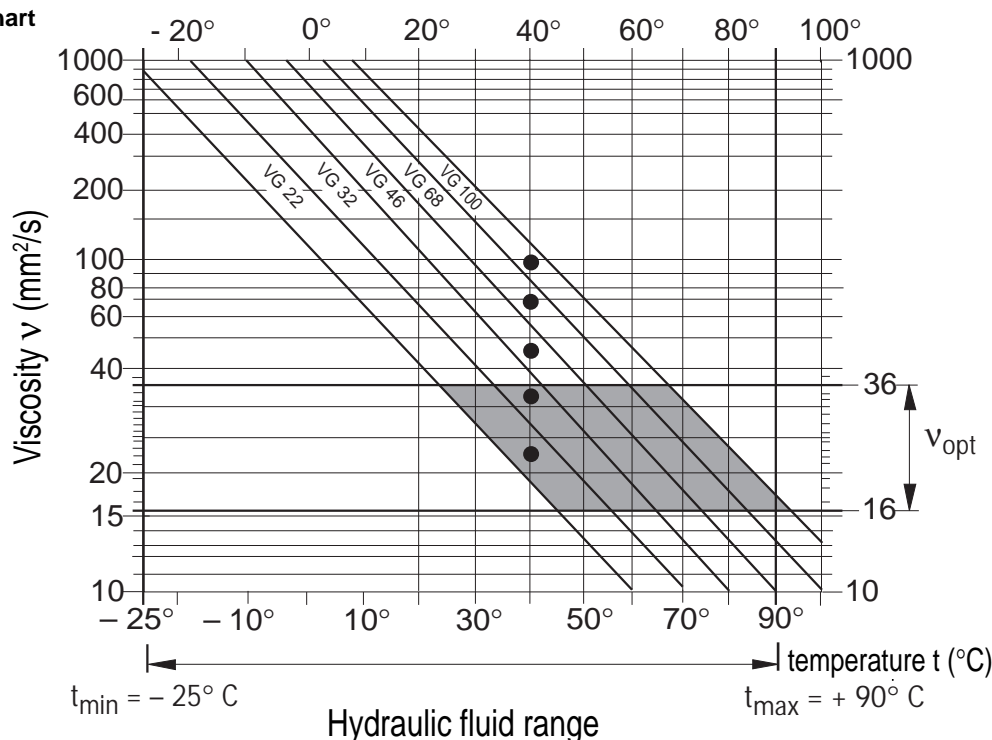
briefly on cold start.

Temperature range (see selection chart)

$$t_{\text{min}} = -25^\circ \text{C}$$

$$t_{\text{max}} = +90^\circ \text{C}$$

Selection chart



Comment on selecting hydraulic fluid

To select the correct hydraulic fluid it is necessary to know the operating temperature in the tank (open circuit) in relation to ambient temperature.

The hydraulic fluid must be selected in such a way that service viscosity lies within the optimum range (v_{opt}) for the operating temperature span, see shaded area in the chart.

We recommend selecting the next higher viscosity class in each case.

Example: An ambient temperature of X °C will produce an operating temperature in the tank of 60 °C. Given the optimum service viscosity range (v_{opt} ; shaded area), this will require viscosity classes VG 46 or VG 68; class to select: VG 68.

Note: The leakage oil temperature, affected by pressure and speed, is always higher than tank temperature. However, temperature must not exceed 90 °C anywhere in the system.

Please contact us if it is not possible to meet the above conditions due to extreme service parameters or high ambient temperature.

Hydraulic fluid filtering at the axial piston pump

To ensure operational reliability, the service fluid must conform to at least purity class

9 to NAS 1638

18/15 to ISO/DIS 4406.

Technical specifications

Service pressure range, input

Absolute pressure at port S

$p_{abs \min}$ _____ 0.8 bar
 $p_{abs \max}$ _____ 5 bar

Service pressure range, output

Pressure at port B

Nominal pressure p_N _____ 250 bar
 Peak pressure p_{\max} _____ 315 bar
 (pressures to DIN 24312)

Direction of flow

S to B.

Leakage fluid pressure

Maximum permissible leakage fluid pressure (at port L, L_1):

$p_{L \max}$ _____ 2 bar absolut
 Always given with end-face leakage oil bore allowing unrestricted drainage into transmission housing.

Pressure at lubrication oil port G:

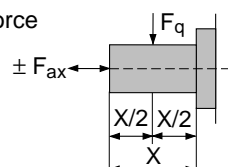
 $p_{abs \min} = 1 \text{ bar}$

Table of values (theoretical values, ignoring η_{mh} and η_v ; values rounded)

Size			45	63
Displacement volume		$V_{g \max}$ cm ³	45	63
Max. speed *	at $V_{g \max}$	$n_{o \max}$ rpm	2750	2600
Max. volumetric flow (delivery)	at $n_{o \max}$	$q_{Vo \max}$ L/min	123	163
Max. power ($\Delta p = 250 \text{ bar}$)	at $n_{o \max}$	$P_{o \max}$ kW	52	68
Max. torque ($\Delta p = 250 \text{ bar}$)	at $V_{g \max}$	T_{\max} Nm	181	250
Moment of inertia around drive axle		J kgm ²	0.0047	0.0056
Fluid capacity		L	0.6	0.8
Mass (without fluid)		m kg	27.2	28.5
Permissible load on drive shaft: max. permissible axial force		$F_{ax \max}$ N	1500	1500
max. permissible lateral force		$F_{q \max}$ N	1500	1500

*intermittently 3000 rpm

Application of force



Calculating size

Volumetric flow	$q_v = \frac{V_g \cdot n \cdot \eta_v}{1000}$	[L/min]	V_g = geometric displacement volume [cm ³] per revolution
Drive torque	$T = \frac{1.59 \cdot V_g \cdot \Delta p}{100 \cdot \eta_{mh}}$	[Nm]	Δp = pressure differential [bar] n = speed [rpm] η_v = volumetric efficiency
Drive power	$P = \frac{2\pi \cdot T \cdot n}{60000} = \frac{T \cdot n}{9549} = \frac{q_v \cdot \Delta p}{600 \cdot \eta_t}$	[kW]	η_{mh} = mechanical-hydraulic efficiency η_t = total efficiency ($\eta_t = \eta_v \cdot \eta_{mh}$)

Characteristic curves for pump with pressure controller DR

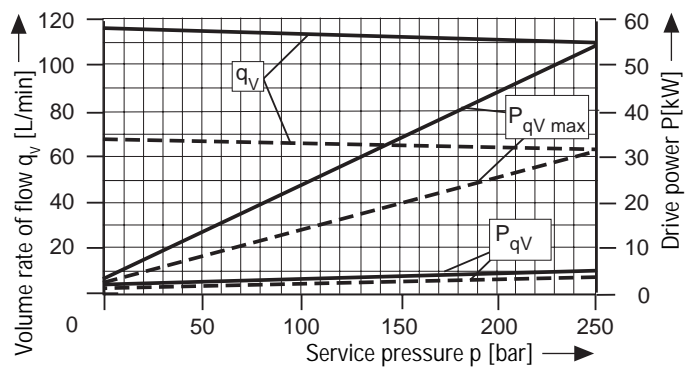
Drive power and delivery rate (volumetric flow)

(service fluid: hydraulic oil ISO VG 46 DIN 51519, $t = 50\text{ }^{\circ}\text{C}$)

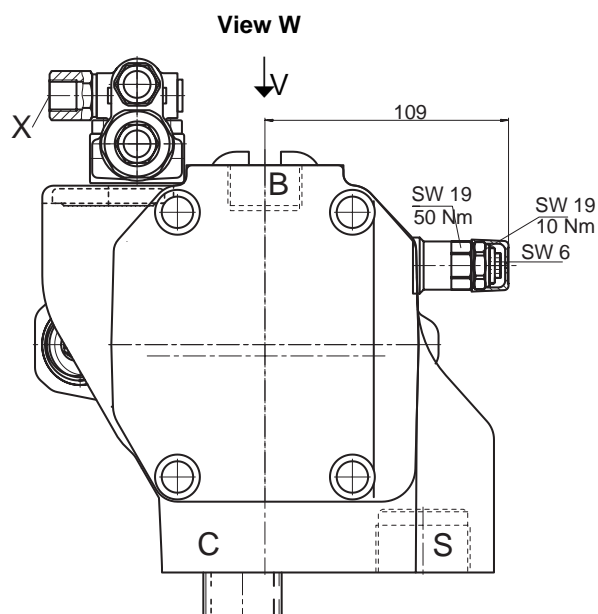
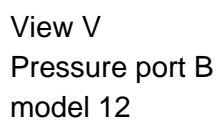
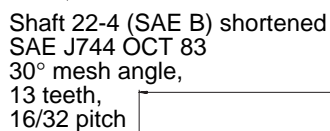
Size 45

--- $n = 1500\text{ rpm}$

— $n = 2600\text{ rpm}$



Type A10CO 45 DR
DFR
DFR1 /52 R -V U C 07 H00 2 D
DRG 12 F

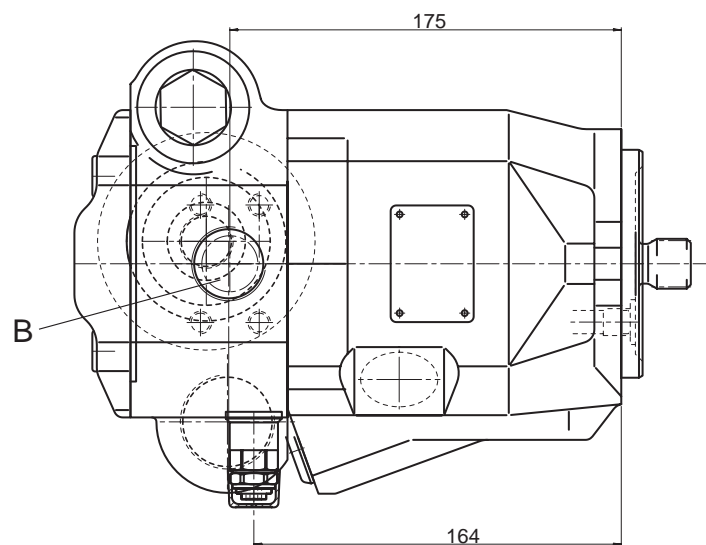
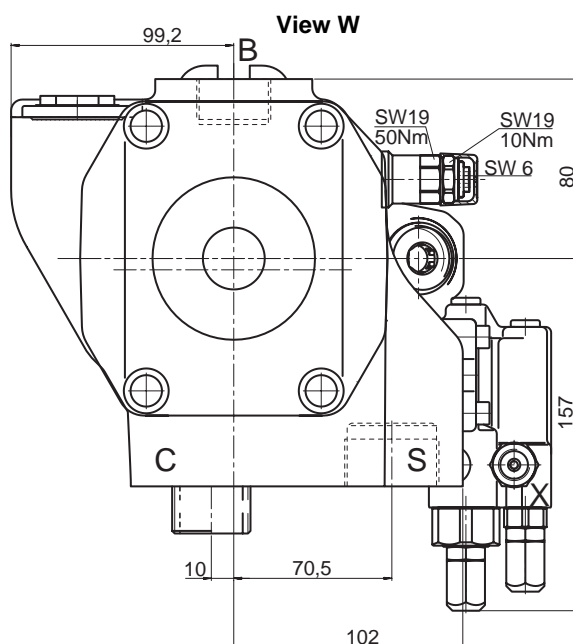
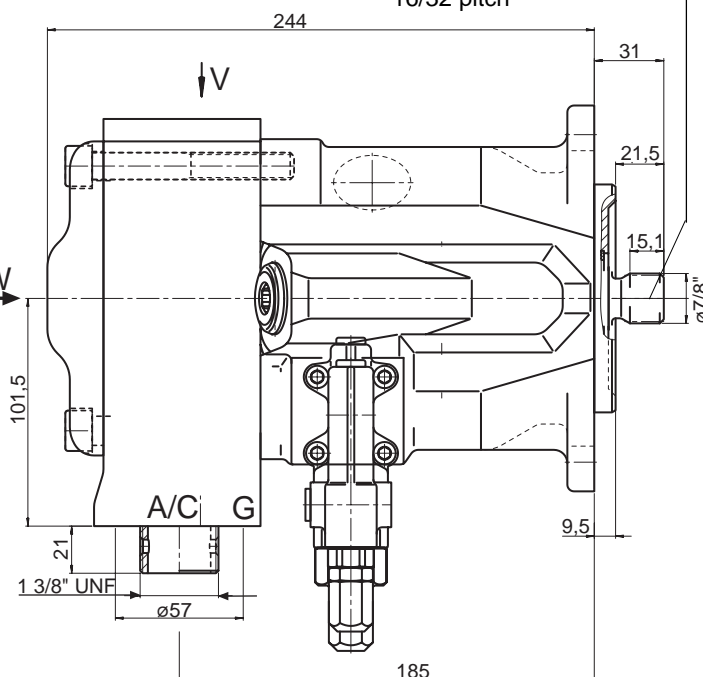
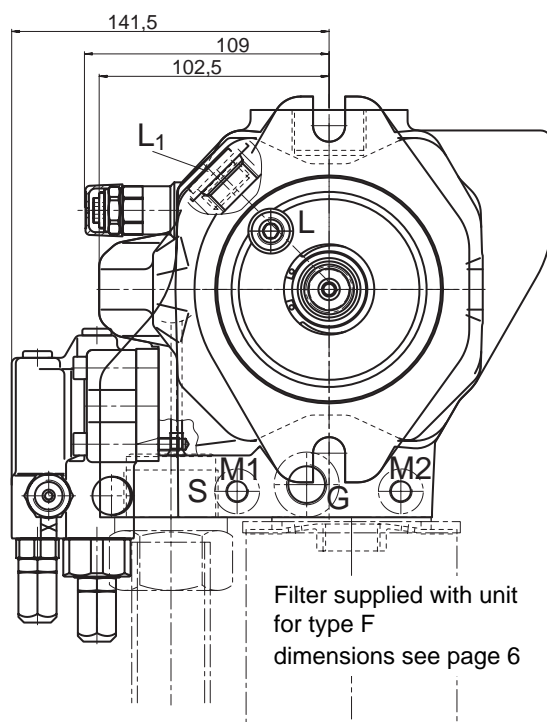


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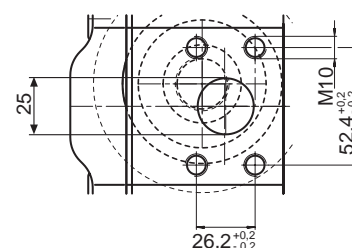
Unit dimensions size 45, anti-clockwise direction of rotation

Type A10CO 45 DR
DFR
DFR1 /52 L -V U C 07 H00 2 D
DRG 12 F

Shaft 22-4 (SAE B) shortened
SAE J744 OCT 83
30° mesh angle,
13 teeth,
16/32 pitch



View V
Pressure port B, model 12

**Ports**

B Pressure port
S Suction port
A/C Filter port, feed pump
to DIN 71457-A1,2

L Case drain port
L₁ Case drain port
X Control pressure port

with pressure controller DR – X port closed

G Transmission lubrication port
M₁, M₂ Measurement ports

Model 07

M33x2
M48x2
1 3/8" UNF-2B

M12x1,5 integrated
7/8-14UNF-2B

M14x1,5

M18x1,5

M10x1

Model 12

SAE 1" (standard pressure)
M48x2
1 3/8" UNF-2B

M12x1,5 integrated
7/8-14UNF-2B

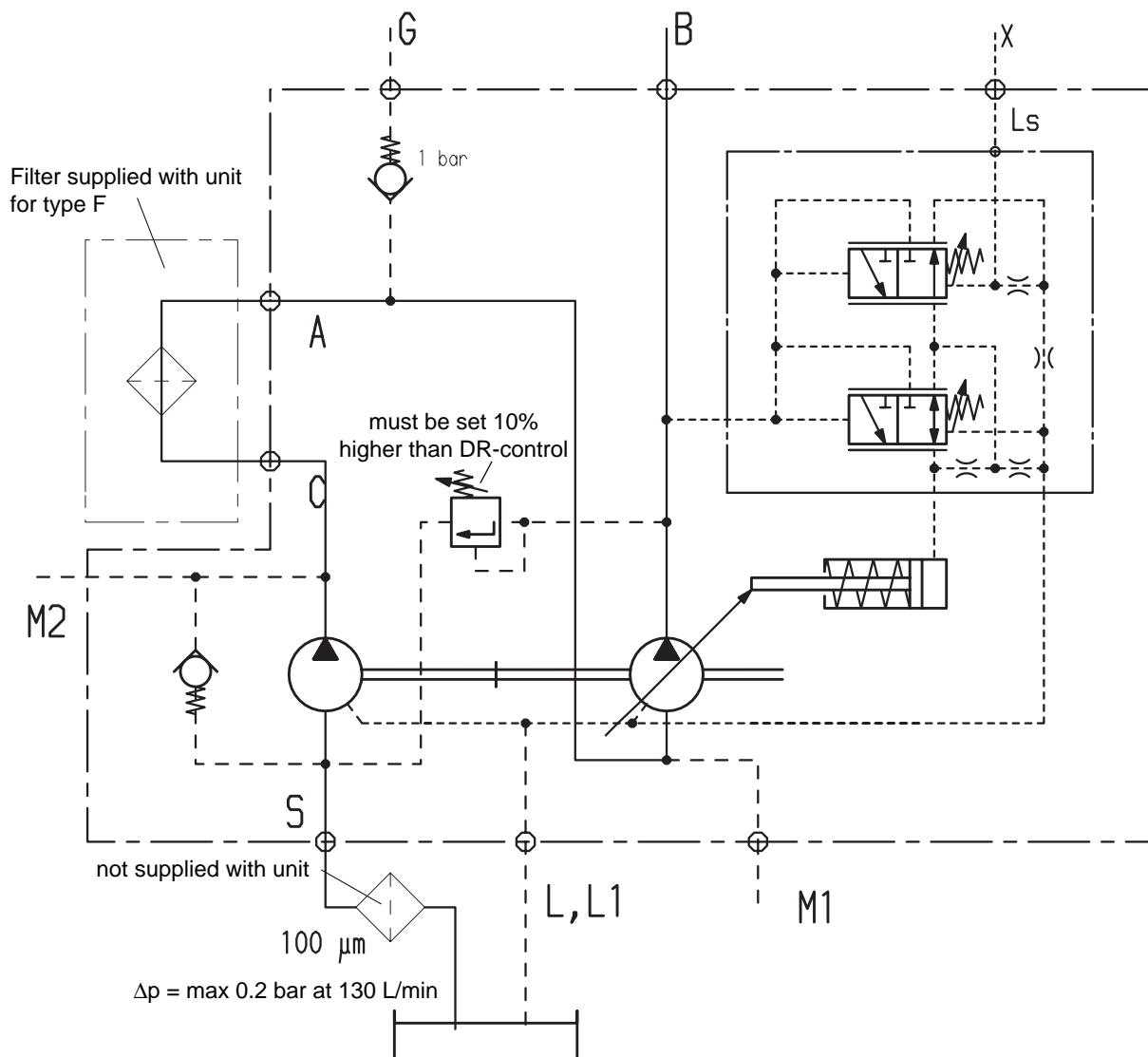
M14x1,5

M18x1,5

M10x1

Circuit diagram

Type A10CO 45 DFR /52 X- V U C XX H00 2D
F



Ports

- B Pressure port
- S Suction port
- A/C Filter port, feed pump
- L, L₁ Case drain ports
- X Control pressure port

with pressure controller DR – X port closed

- G Transmission lubrication port
- M₁, M₂ Measurement ports